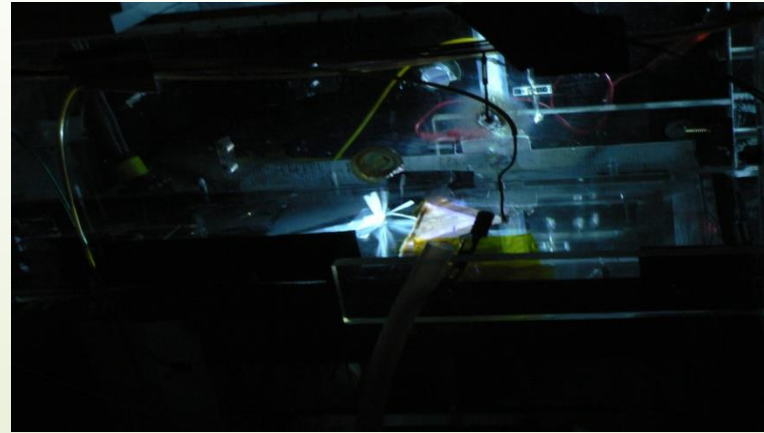


# Feasibility of Self Powered Actuation for Flow, Separation and Vibration Control



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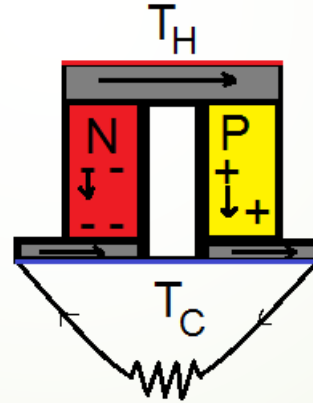
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<sup>[3]</sup> Research Aerospace Engineer, NASA GRC, contact author

# Thermoelectric Generators

- TEGs make use of the Seebeck effect - if two dissimilar metals are connected in such a way that they form a circuit, and if the junctions of those two metals are held at different temperatures, an electric potential is generated between the hot and cold junctions.

$$P = \eta Q_{TEG} = \eta \frac{T_{HJ} - T_{CJ}}{R_{TEG}}$$

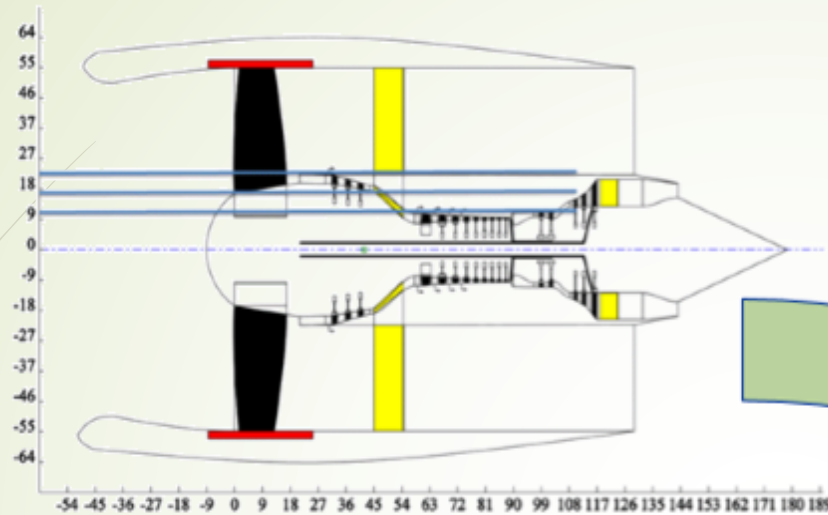


$$\eta = \frac{\Delta T}{T_h} \frac{\sqrt{1 + Z\bar{T}} - 1}{\sqrt{1 + Z\bar{T}} + T_c/T_h}$$

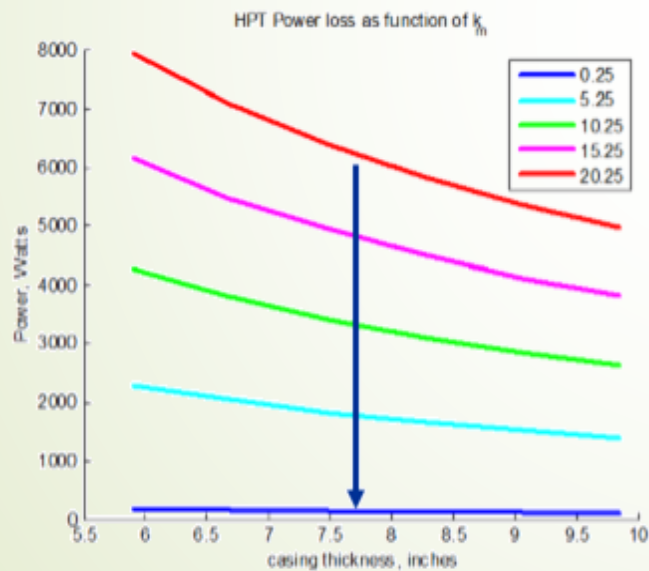
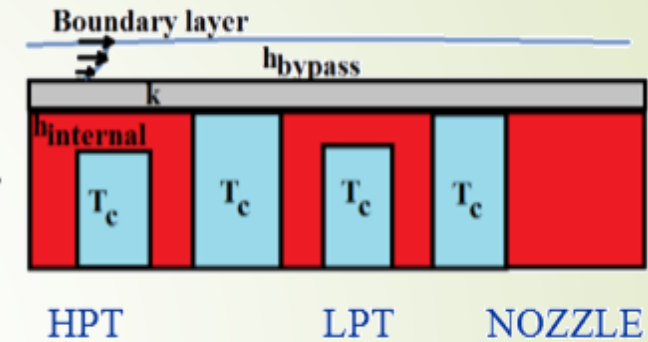
$$Z\bar{T} = \frac{eS^2\bar{T}}{k} \quad \bar{T} = \frac{T_H + T_C}{2}$$

# Harvesting in a Jet Engine

- In a gas turbine engine heat is lost radially through conduction in the casing walls and through the exhaust nozzle.
- Energy harvested from this temperature difference can be stored in a capacitor or battery as part of a hybrid electric engine system. (Out of scope for this talk)
- Large temperature differences ( $\sim 800\text{K}$ ) also exist between the blade outer surfaces (air that's been through combustion) and their internal surfaces (exposed to cooling air.)
- This heat transfer is not considered waste heat because the cooling air is returned to the main flow path and turns downstream blade rows.
- The benefit then of harvesting energy from the temperature difference across the thickness of a turbine blade is a local power source for active flow control, actuation or sensors.



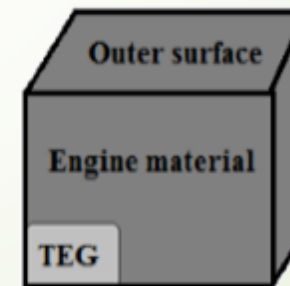
Idealized model of Turbine and Nozzle



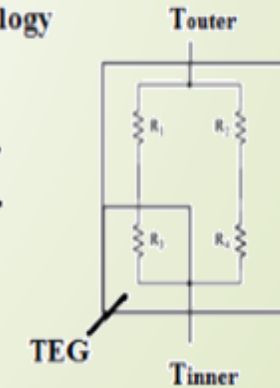
$$100W$$

$$\Delta T_{\text{metal}} = +10K$$

Section of engine skin with embedded TEG



Thermal-electrical resistance analogy



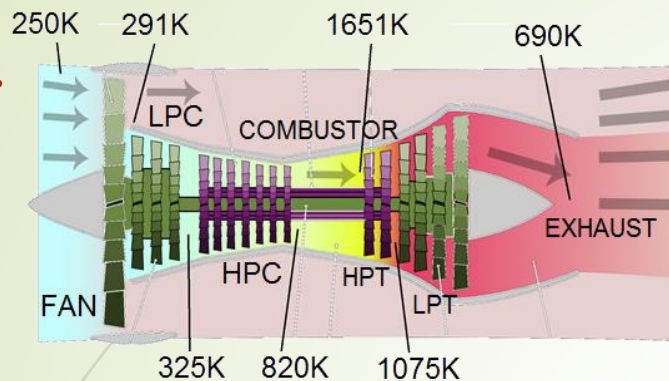


Figure 1 Cross section of a turbofan engine with representative temperatures obtained from NPSS [1] at cruise (Mach 0.85, 35k ft) for a high bypass ratio engine.

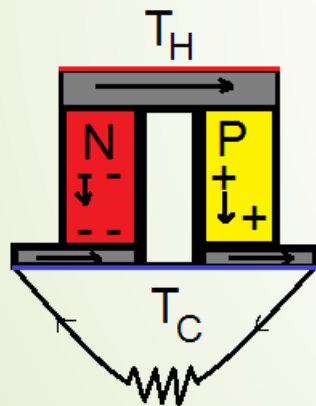


Figure 4. Schematic of a TEG (adapted from [2]).



Figure 2. Turbine vanes with cooling holes

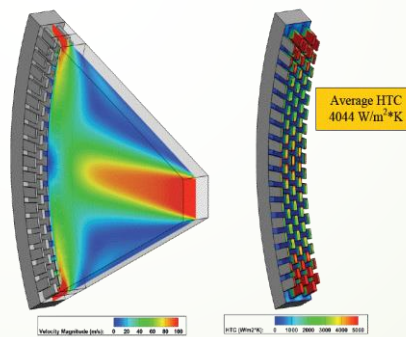


Figure 5. Velocity magnitude slice and blade interior surface heat transfer coefficient for an impingement region with strip fins and  $w = 2$  mil.

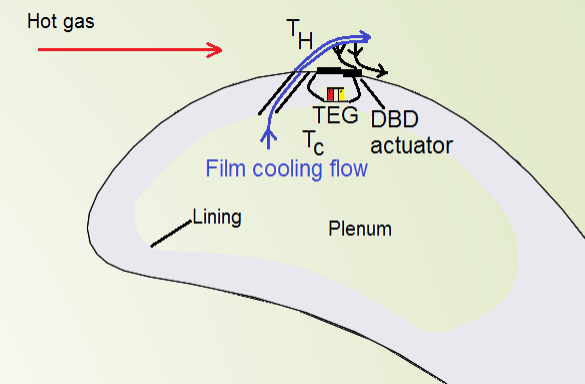


Figure 3 Self-powered DBD actuator system showing control of film cooling jet using a DBD actuator powered by energy harvested from an embedded TEG.

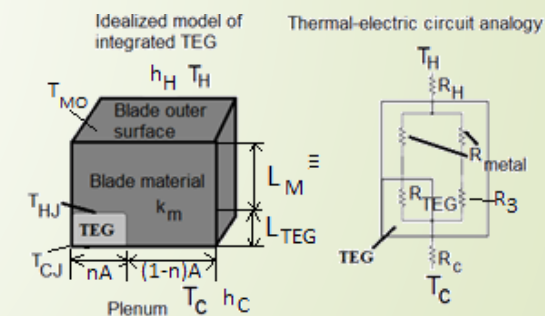


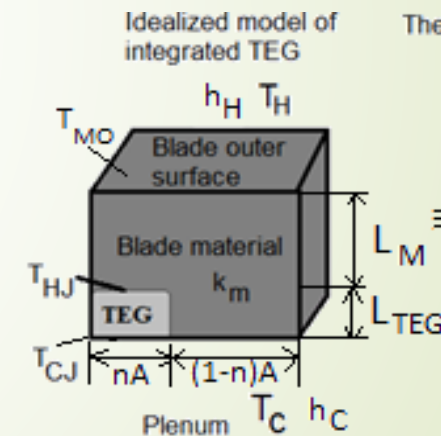
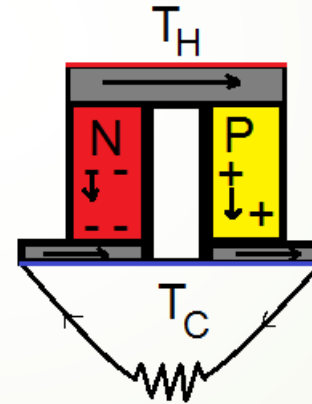
Figure 6. Left - section of blade with TEG embedded. Right - thermal circuit for TEG embedded in blade.



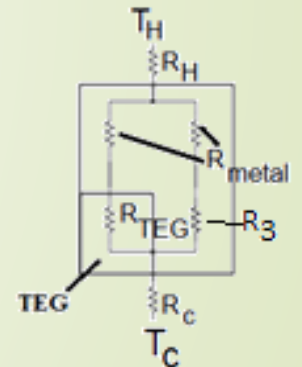
# Power Output Simplified

$$P = \frac{\eta(T_H - T_C)}{R_{TEG}} \left\{ 1 - \frac{1}{1 + \frac{h_C h_H L_{TEG}}{\left(1 - n + n \frac{k_{TEG}}{k_m}\right) [k_m(h_C + h_H) + L_m h_C h_H]}} \right\}$$

$$\dot{P}_{\sim EG} \left\{ 1 - \frac{1}{1 + HT} \right\}$$

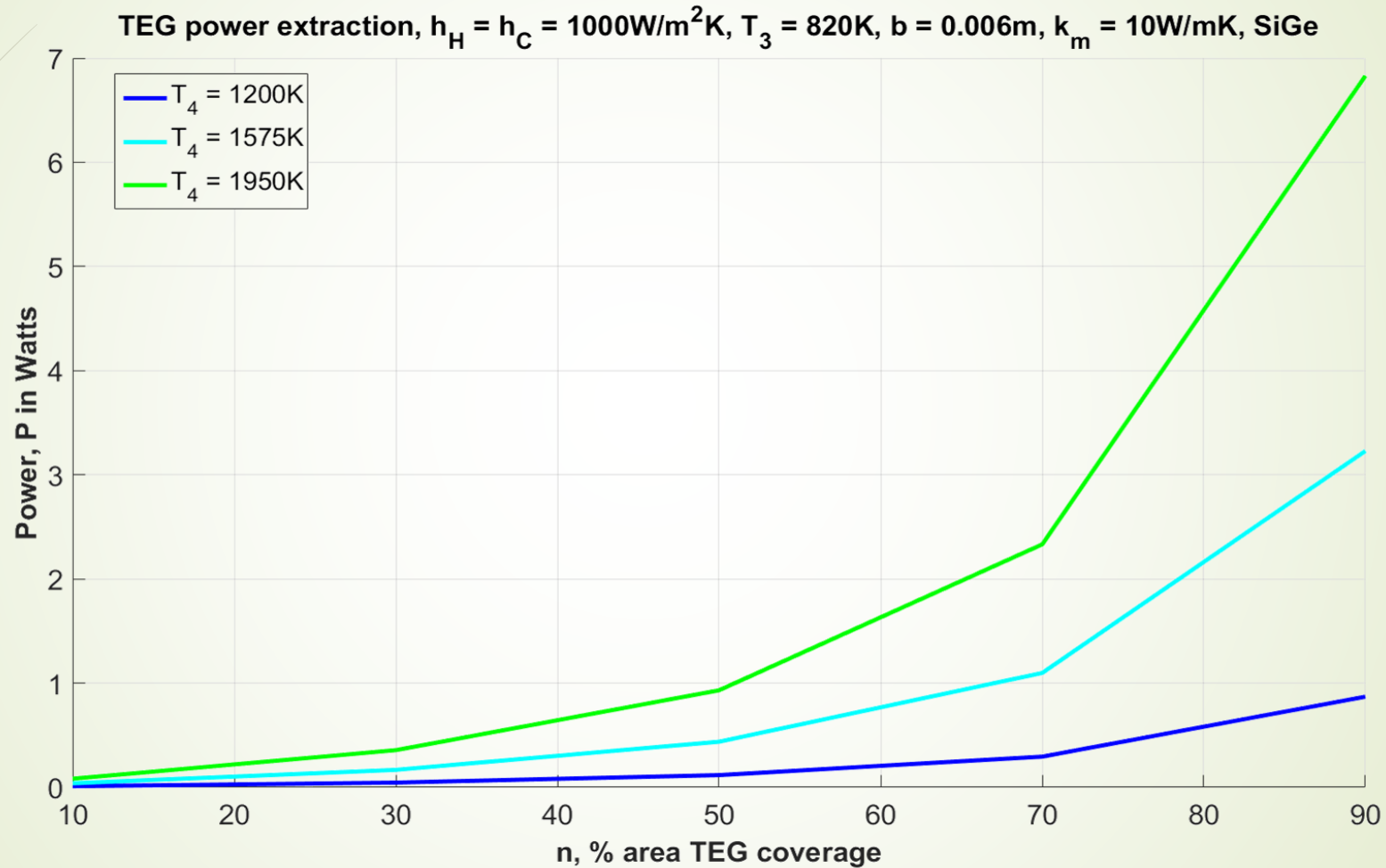


Thermal-electric circuit analogy

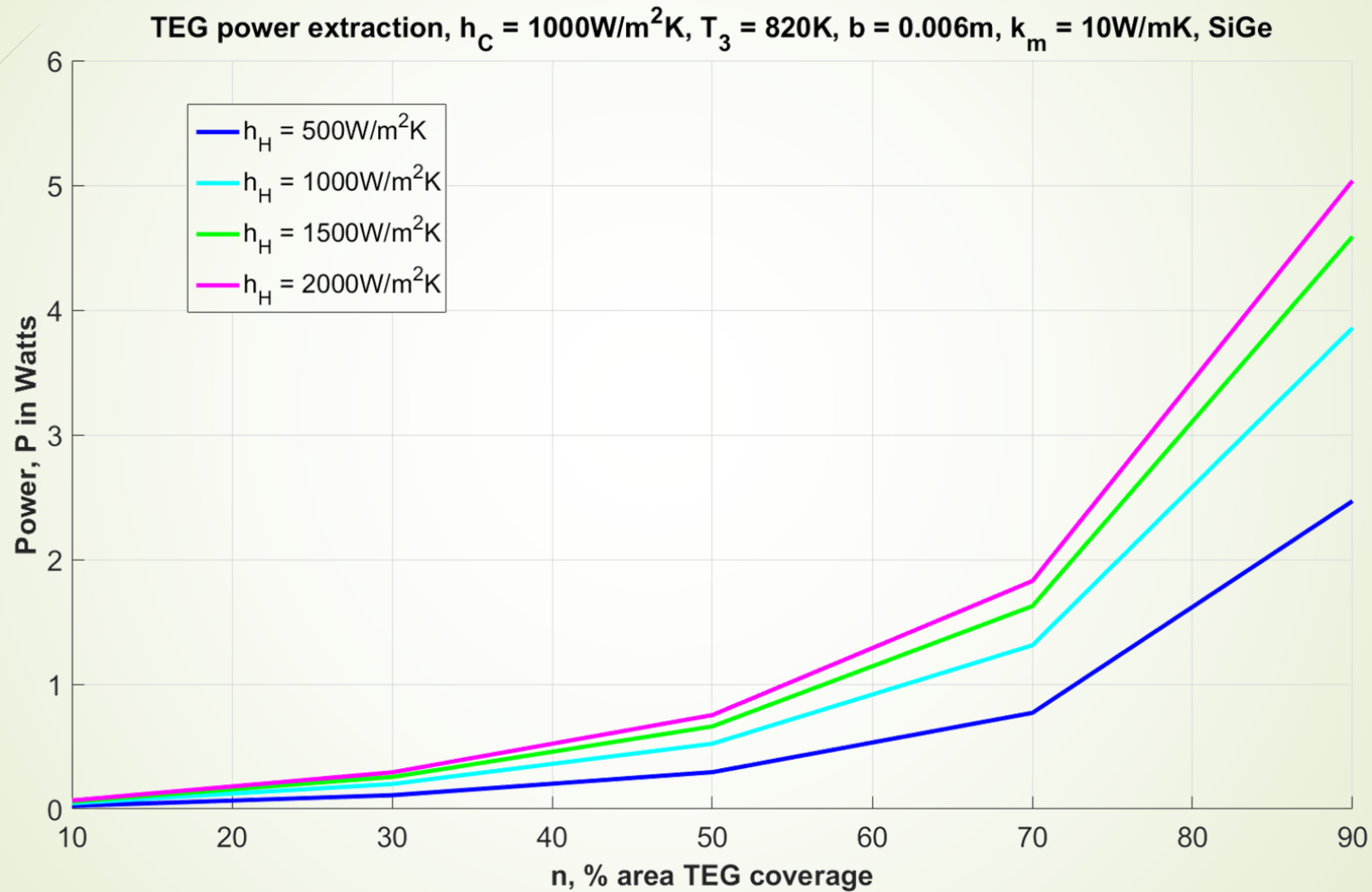


$$G = \frac{\eta}{R_{TEG}}, E = T_H - T_C$$

$$F = \underbrace{\frac{h_C h_H}{[k_m(h_C + h_H)]}}_H \underbrace{\frac{L_{TEG}}{(1 - n)}}_T$$

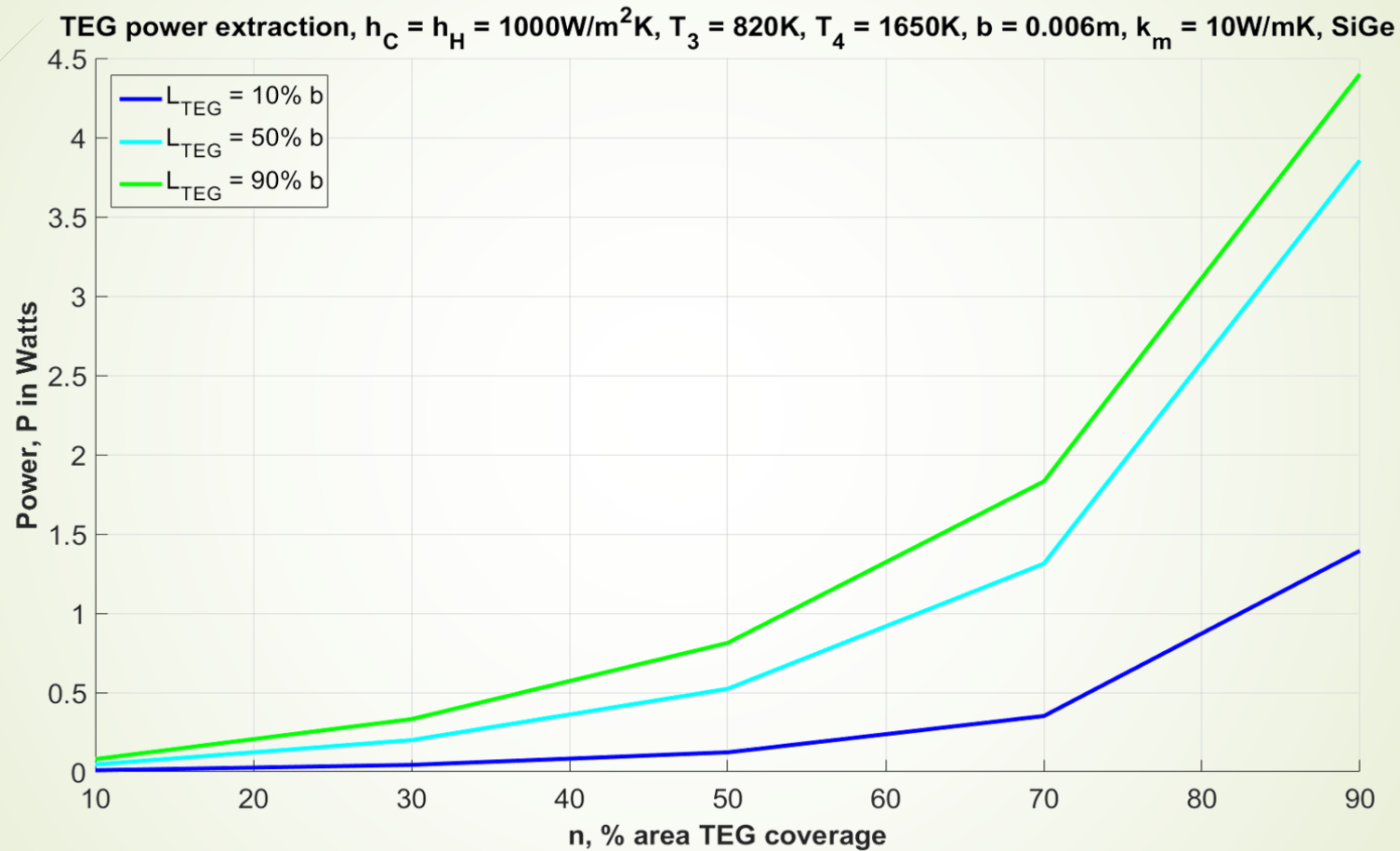


**Figure 11. Power output by TEG for various hot gas temperatures,  $T_4$  as a function of % plenum surface area used for heat extraction.**

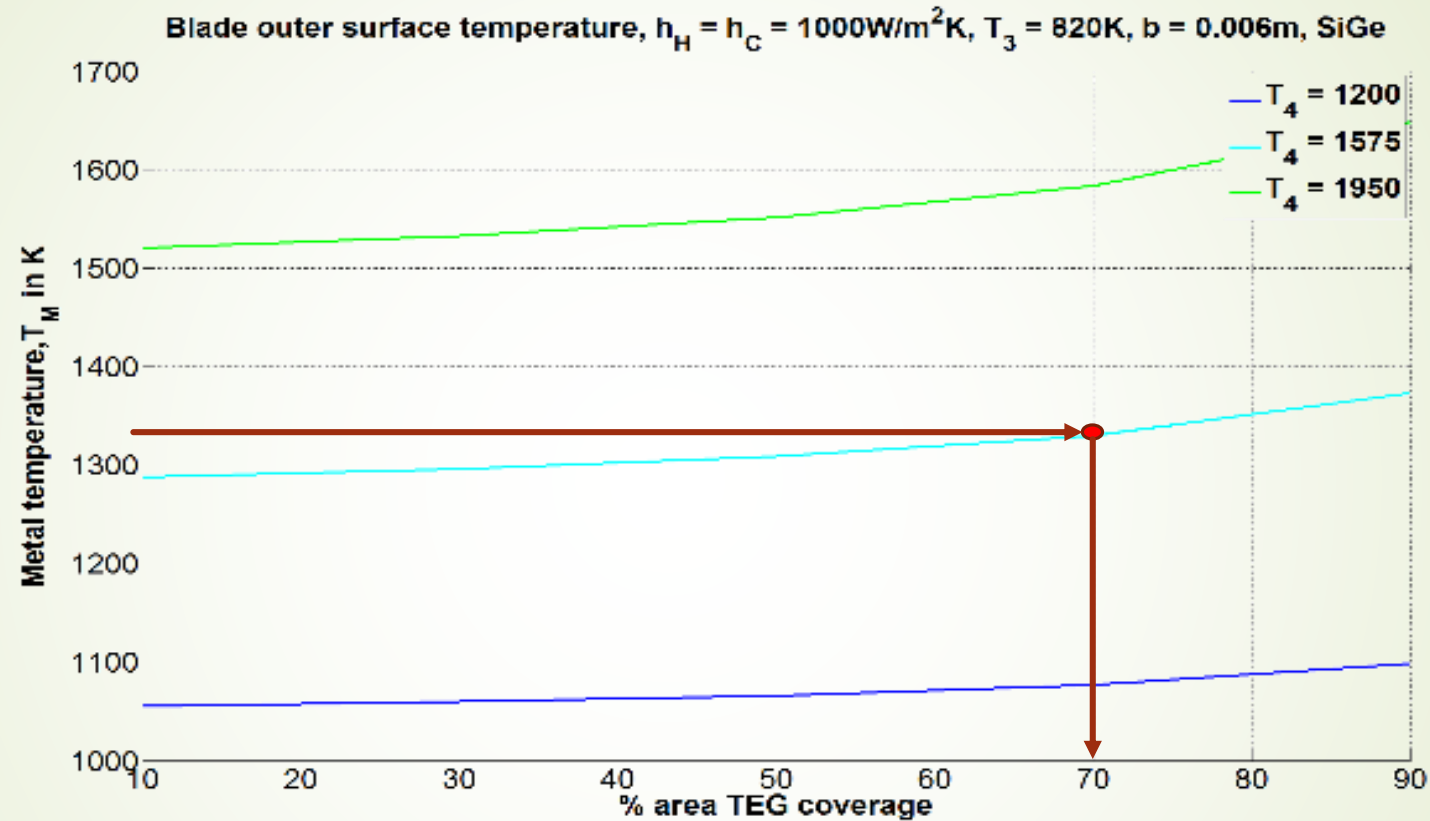


**Figure 13. TEG power as a function of % plenum surface area used for heat extraction and hot gas heat transfer coefficient.**





**Figure 16. TEG power as a function of TEG thickness, and % plenum surface area used for heat extraction.**

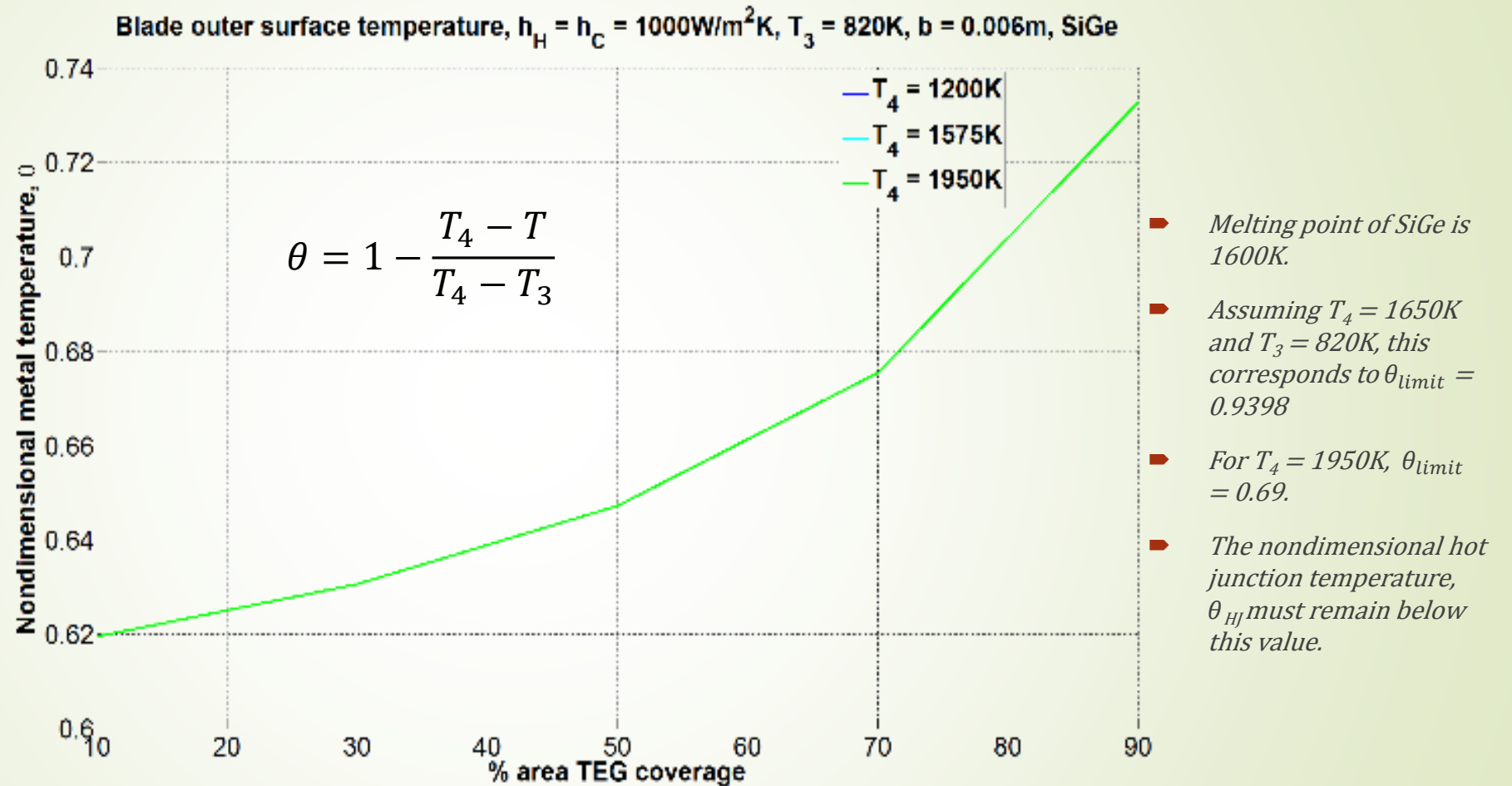


- Blade thermal limit =  $1350 \text{ K}$  ( $\sim 2000^\circ \text{F}$ )
- $T_4 = 1575 \text{ K}$  ( $2375^\circ \text{F}$ ).
- may use less than 70% of plenum surface for TEG

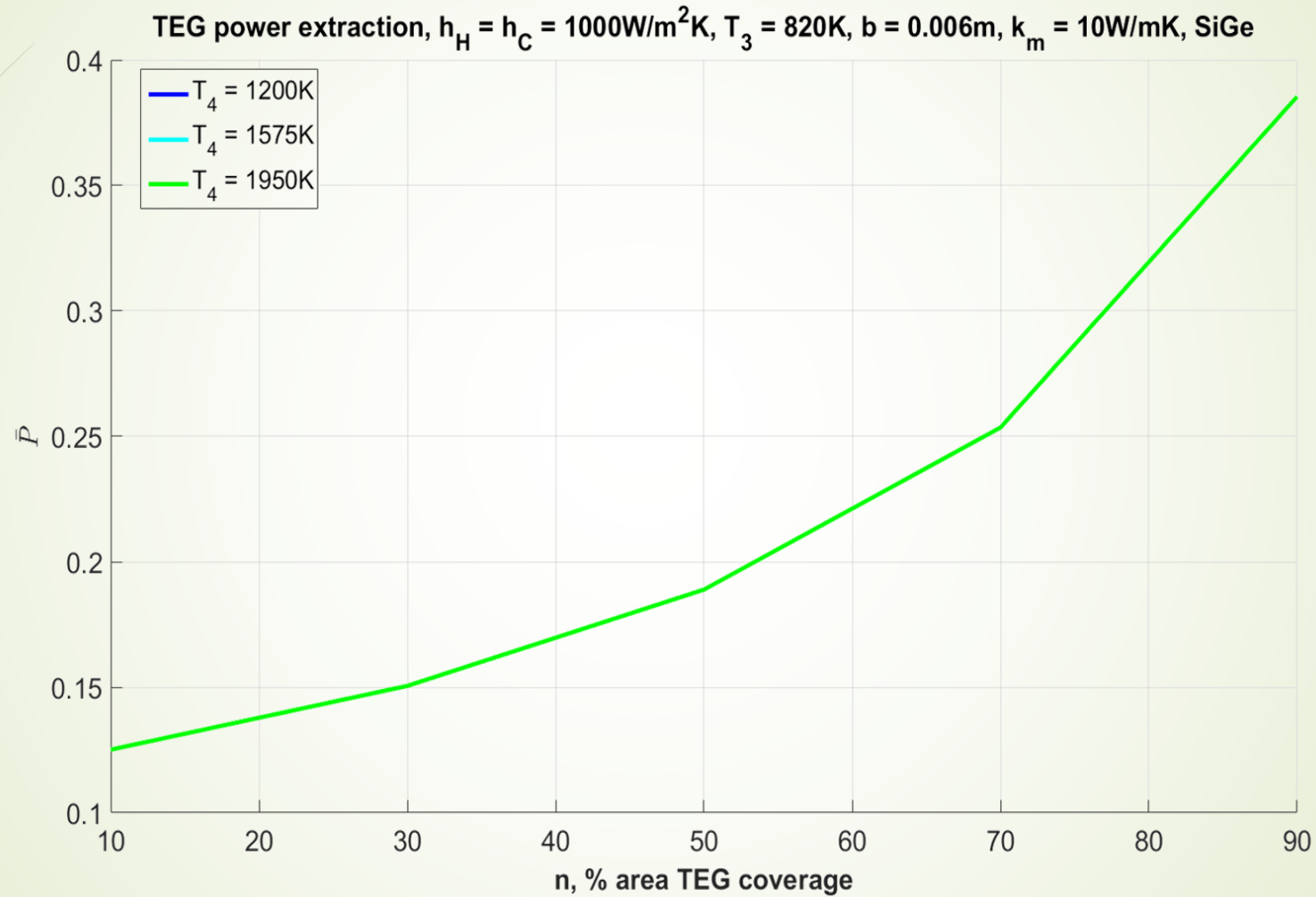
**Figure 18. Blade outer surface temperature as a function of % plenum surface area used for heat extraction.**

# Trying to collapse things...

- $\bar{P} = \frac{P}{\dot{q} \cdot \Delta T_{43}}$  (Power mainly depends on  $\Delta T_{43}$  for a given TEG)
- $\theta = 1 - \frac{T_4 - T}{T_4 - T_3}$
- $\theta = 0$  corresponds to  $T_3$
- $\theta = 1$  corresponds to  $T_4$



**Figure 13. Blade outer surface nondimensional temperature as a function of % plenum surface area used for heat extraction.**



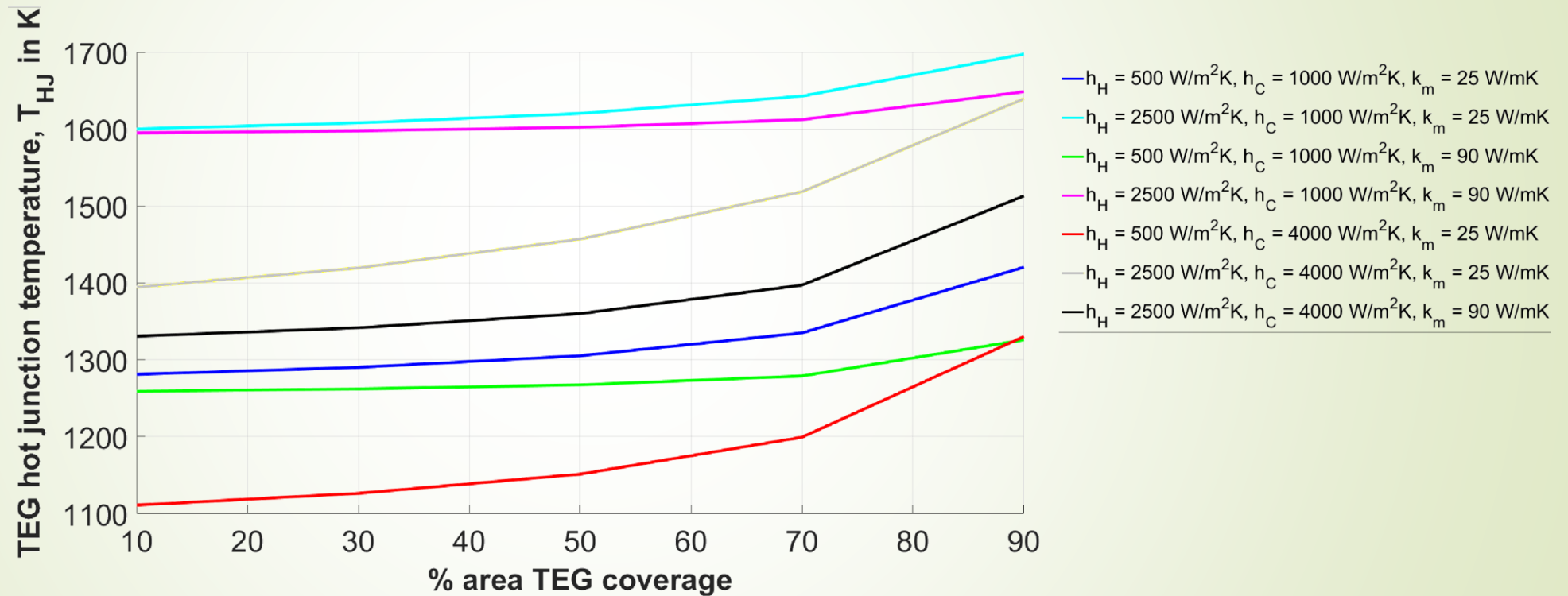
**Figure 12. TEG corrected power as a function of % plenum surface area used for heat extraction.**



# Applying to realistic conditions...

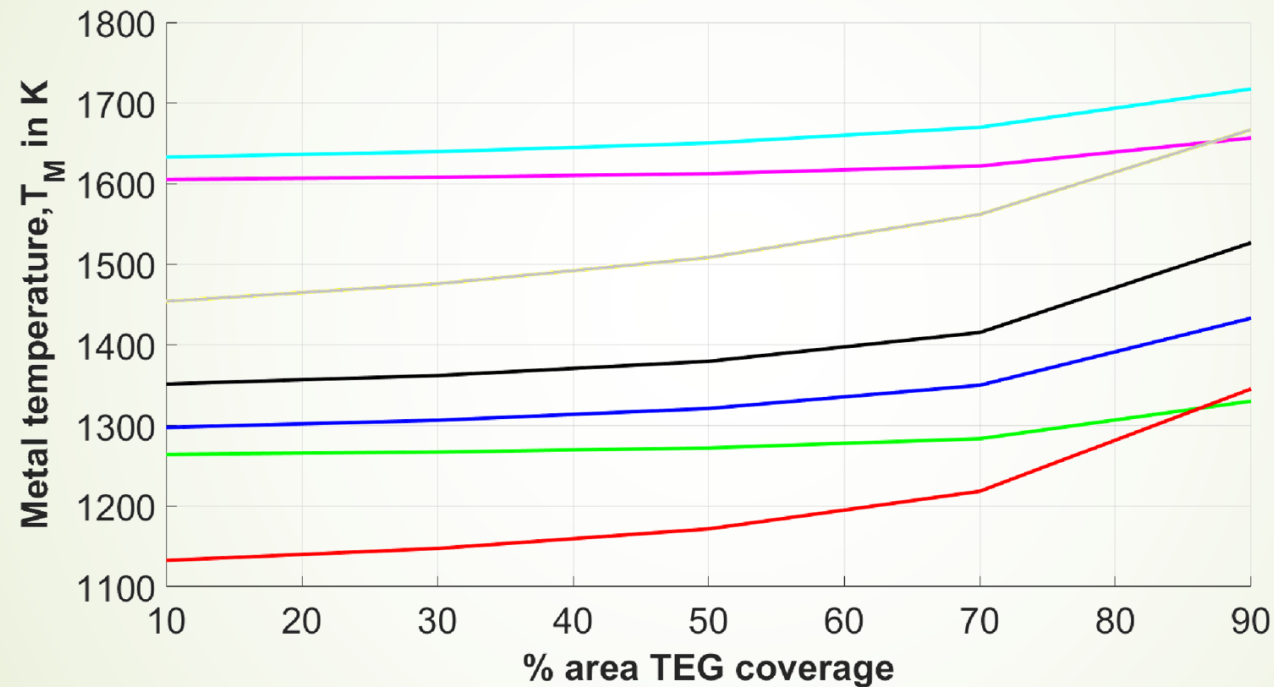
- At take-off,  $T_4 = 1850\text{K}$  and  $T_3 = 950\text{K}$
- TEG is SiGe (high temperature limit)
- Blade thickness =  $0.006\text{m}$  ( $\sim 0.23\text{in}$ )
- Usable plenum surface area =  $.000322\text{m}^2$  ( $0.5\text{sq. in.}$ )
- TEG thickness = 50% of the blade thickness
- 2 values of hot gas heat transfer coefficient,  $h_H$  considered correspond to 2 extreme cases
  - $500\text{ W/m}^2\text{K}$  (trailing edge regions, no film cooling)
  - $2500\text{ W/m}^2\text{K}$  (leading edge of vane/blade).
- Similarly for the plenum,
  - $h_C = 1000\text{ W/m}^2\text{K}$ ,  $h_C = 4000\text{ W/m}^2\text{K}$  (simple ribs and microchannels respectively)
- Thermal conductivities -  $25\text{ W/mK}$ ,  $90\text{ W/mK}$  (single crystal superalloys and conventionally cast superalloys)

# TEG hot junction temperature at T.O.



a. TEG hot junction temperature,  $T_3 = 950\text{K}$ ,  $T_4 = 1850\text{K}$ ,  $b = 0.006\text{m}$ , SiGe

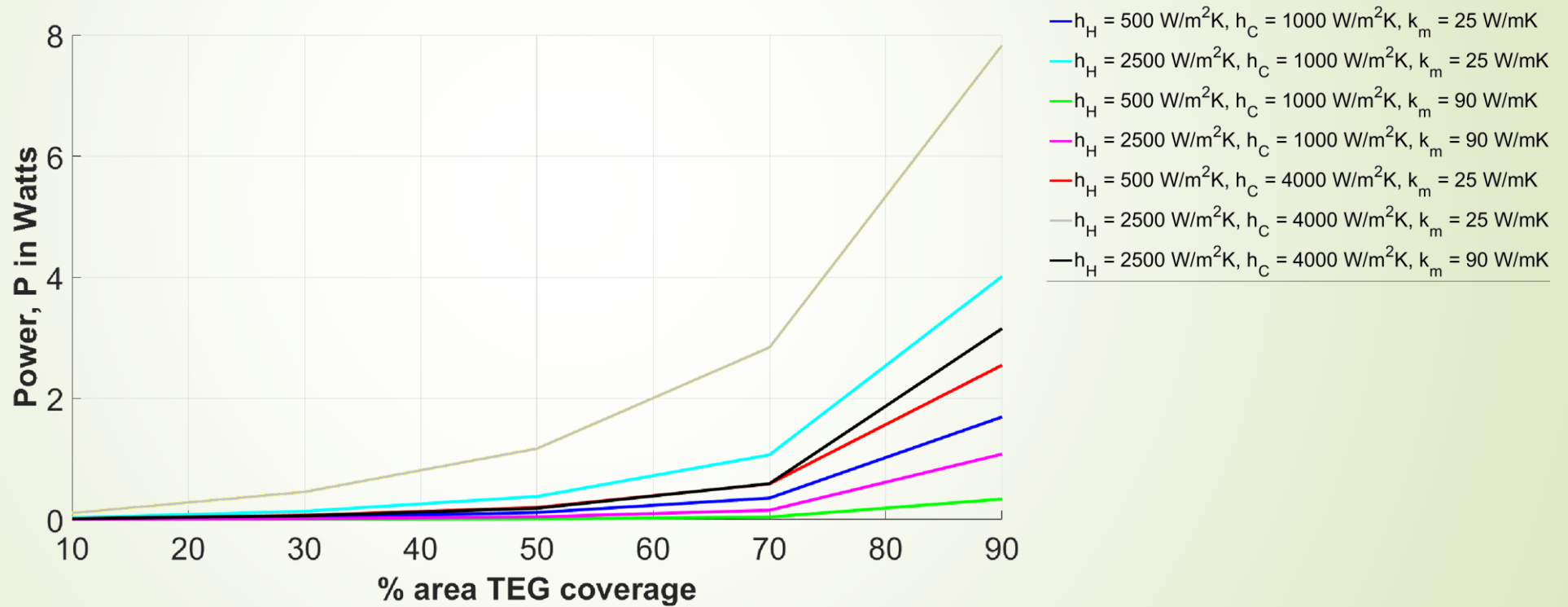
# Outer blade temperature at T.O.



b. Blade outer surface temperature,  $T_3 = 950\text{K}$ ,  $T_4 = 1850\text{K}$ ,  $b = 0.006\text{m}$ , SiGe

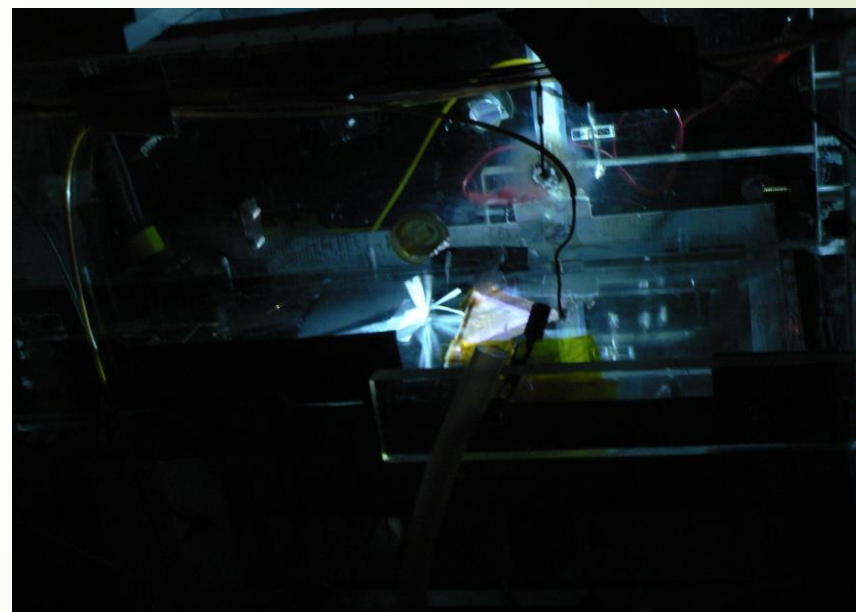
- $h_H = 500 \text{ W/m}^2\text{K}$ ,  $h_C = 1000 \text{ W/m}^2\text{K}$ ,  $k_m = 25 \text{ W/mK}$
- $h_H = 2500 \text{ W/m}^2\text{K}$ ,  $h_C = 1000 \text{ W/m}^2\text{K}$ ,  $k_m = 25 \text{ W/mK}$
- $h_H = 500 \text{ W/m}^2\text{K}$ ,  $h_C = 1000 \text{ W/m}^2\text{K}$ ,  $k_m = 90 \text{ W/mK}$
- $h_H = 2500 \text{ W/m}^2\text{K}$ ,  $h_C = 1000 \text{ W/m}^2\text{K}$ ,  $k_m = 90 \text{ W/mK}$
- $h_H = 500 \text{ W/m}^2\text{K}$ ,  $h_C = 4000 \text{ W/m}^2\text{K}$ ,  $k_m = 25 \text{ W/mK}$
- $h_H = 2500 \text{ W/m}^2\text{K}$ ,  $h_C = 4000 \text{ W/m}^2\text{K}$ ,  $k_m = 25 \text{ W/mK}$
- $h_H = 2500 \text{ W/m}^2\text{K}$ ,  $h_C = 4000 \text{ W/m}^2\text{K}$ ,  $k_m = 90 \text{ W/mK}$

# Power output at T.O.



C. TEG power extraction,  $T_3 = 950K$ ,  $T_4 = 1850K$ ,  $b = 0.006m$ , SiGe

Blades with thermal conductivity of 90W/m.K could generate anywhere from 0.5 - 8W



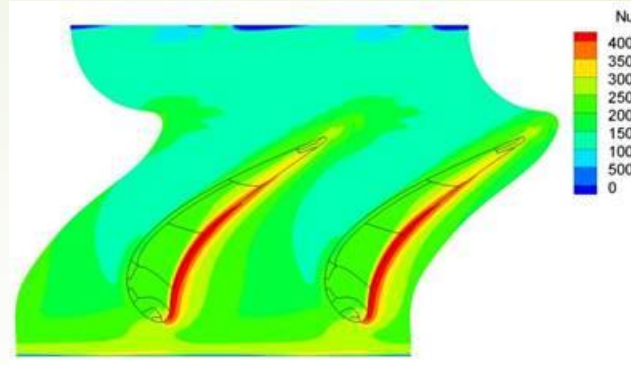


# Conclusions

- *As plenum heat transfer coefficient increases, power output increases*
  - *increase in power with heat transfer coefficient is more apparent as %TEG coverage increases*
- *increasing TEG thickness from 50% to 90% only has significant benefits in the range  $0.3 < n < 0.7$  and is a much smaller benefit than going from 10% to 50%.*
- *increasing TEG layers leads to increased vane outer surface temperature.*

# Conclusions

- There is a tradeoff that can be made between TEG power extraction and technology in the plenum to increase heat transfer coefficient.
- Film cooling effectiveness provides another control parameter (0.1 increase in effectiveness ~ 100K thermal limit increase)
- Lower thermal conductivity blade provides more power but higher metal temperature
- Method to determine TEG material and parameters for integration has been shown



- Nusselt number at casing of HPT was used to get heat transfer coefficient approximations for hot gas path. The heat transfer coefficient in the hot gas path was approximated as  $h = k \cdot Nu / L_{seg}$ . Average Nusselt numbers for HPT (2500), for LPT (1500) and for Nozzle (800) were used. Conductivity of air is based on local temperature.
- Bypass heat transfer coefficient is based on Nusselt number for a flat plate.
- Lamyaa A. El-Gabry, Ali A. Ameri, "COMPUTATION OF UNSTEADY HEAT TRANSFER ON THE CASING OF A TURBINE BLADE: EFFECT OF INLET TOTAL TEMPERATURE WAKE", The 9th International Congress of Fluid Dynamics & Propulsion, Alexandria, Egypt, December 18-21, 2008